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Seminar 21 - Optimizing Design and Controls for Thermal Energy Storage at the Building and Community Scale

Optimizing phase change composite thermal energy storage using the thermal Ragone
framework

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Learning Objectives

- Describe how thermal energy storage can result in a lower levelized cost of storage than battery energy storage
- Demonstrate how additive manufacturing can result in advanced heat exchanger design and manufacturing for thermal energy storage devices
- Explain how thermal power requirements impact the optimal design needs of thermal energy storage heat exchangers
- Explain the benefits of using optimization to inform distribution and control of mixed types of cool thermal storage across a connected community

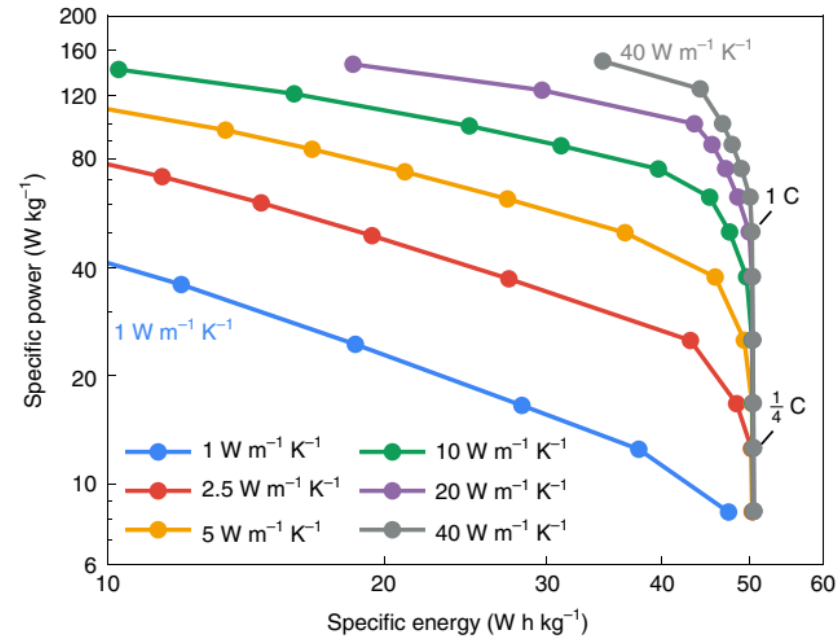
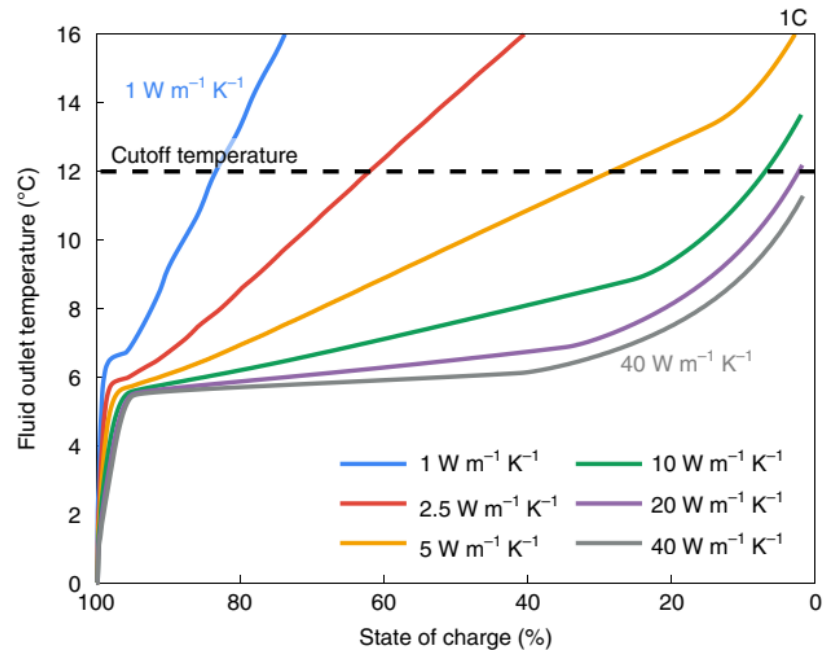
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Outline

- Thermal Ragone Framework
- Design Optimization
- Finite Difference Model and Results
- Approximate Models
- Model Comparisons
- Conclusions

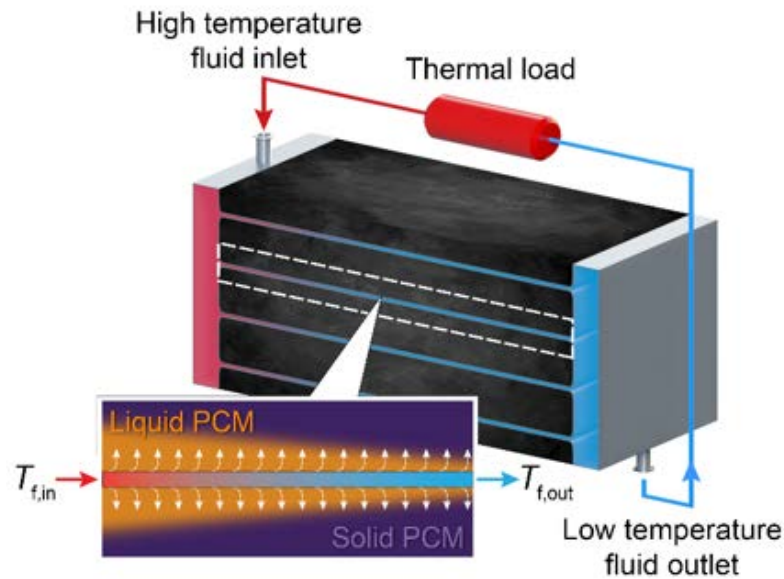
Thermal Ragone Framework



Woods et al. 2021

- Ragone plots have been recently applied to PCM thermal energy storage devices to illustrate the tradeoffs between power requirements and energy storage capacity
- A cutoff temperature can be defined to determine when the useful storage capacity has been depleted
- Material and geometric properties like the PCM thermal conductivity and spacing of the heat transfer fluid channels can have significant impacts on the shape of the Ragone plot for a particular storage device

Device Optimization



Woods et al. 2021

Operational

- Power requirement (C-rate)
- Fluid temperature difference
- Cutoff Temperature
- Initial storage temperature

Geometry

- Heat transfer fluid tube spacing
- Tube thickness
- Porosity of composite additives

Material

- PCM latent heat capacity
- PCM transition temperature
- PCM and composite additive densities
- Thermal conductivities of PCM and composite additives
- Material Costs

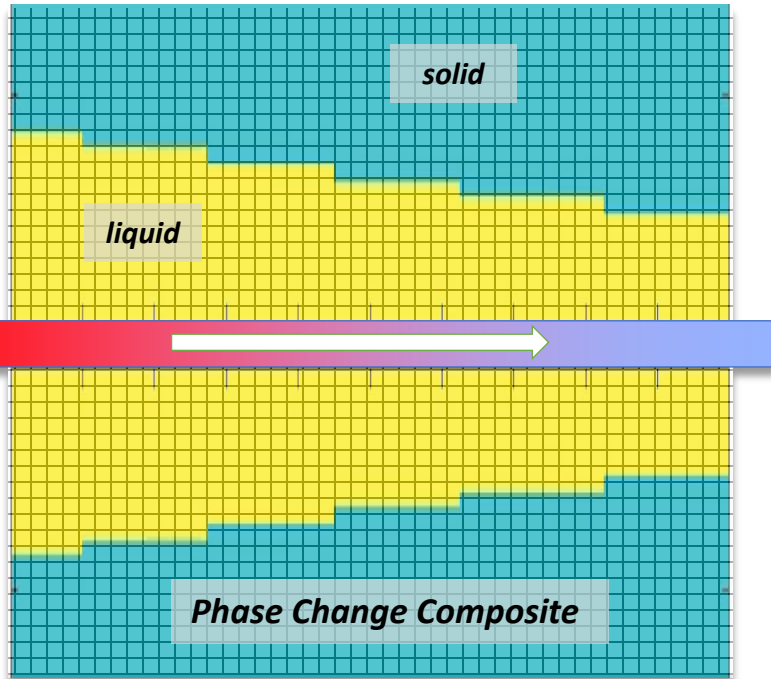
Design Objectives

Maximize Volumetric Energy Density
[kWh/m³]

Minimize Upfront Costs
[\$/kWh_τ]

Minimize Levelized Costs of Storage
[\$/kWh_e]

Finite Difference Model



$$\frac{dh_{ij}}{dt} = \frac{W_{HX}}{M_{ij}} \left[\frac{k_{\parallel} dy}{dx} (T_{i,j-1} + T_{i,j+1} - 2T_{ij}) + \frac{k_{\perp} dx}{dy} (T_{i-1,j} + T_{i+1,j} - 2T_{ij}) \right]$$

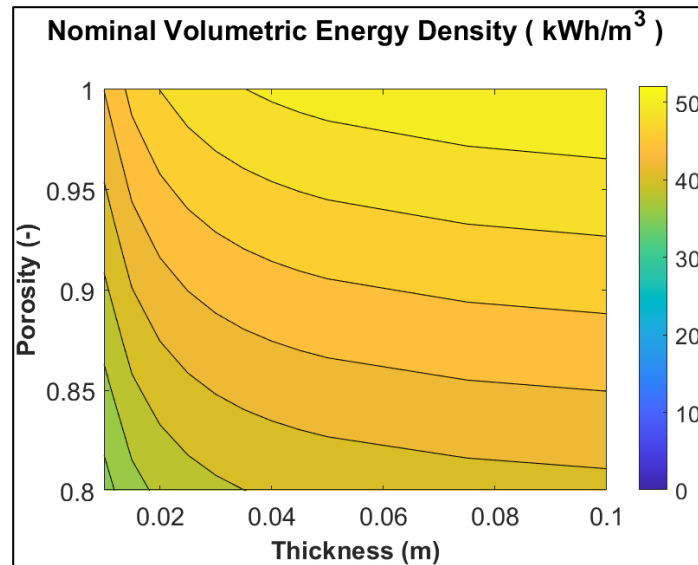
Woods et al. 2021

- A 2D transient numerical model using the finite-difference approach representing a planar thermal energy storage device
- Captures the progression of phase change during the discharge process and the effect that has on the heat transfer fluid outlet temperature
- Assumes the PCM is static, and conduction is the only mode of heat transfer in the composite
- Performed a parametric assessment (>14,000 combinations) to capture the influence of device parameters on the performance of a storage device in a space cooling application.
 - C-rates: 1/6 – 3
 - Transition Temperature: 1°C – 9°C
 - PCM layer thickness: 1cm – 20 cm
 - Porosity of conductivity additives: 80% - 100%

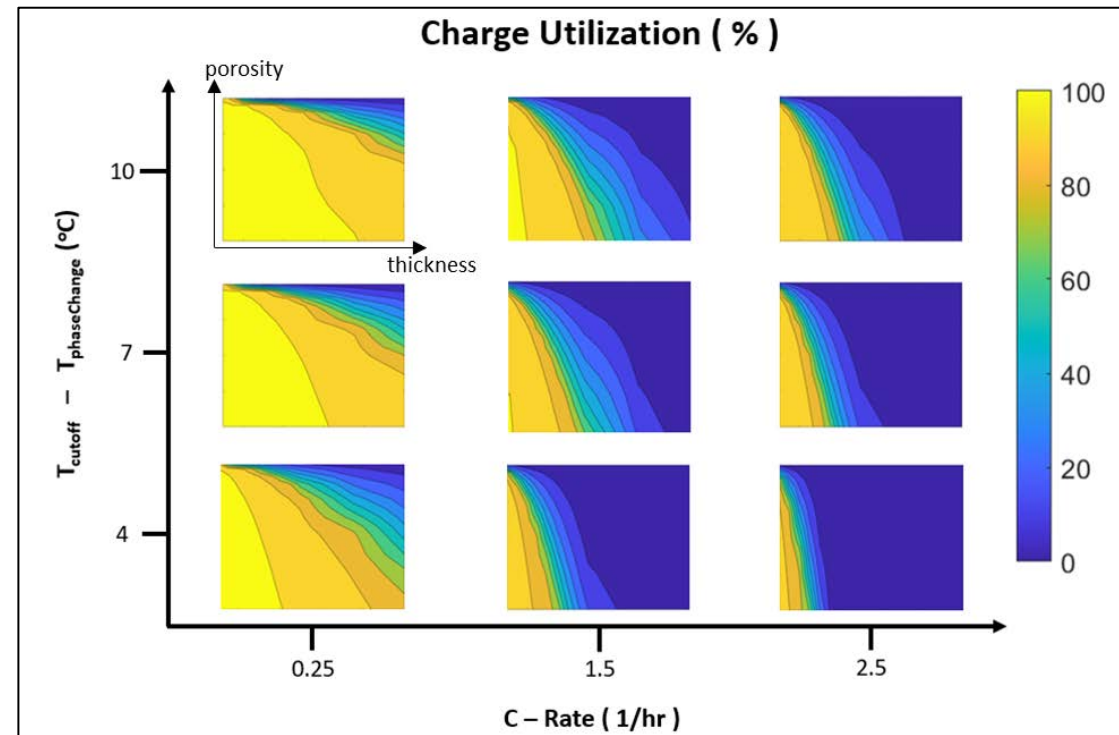
Finite Difference Model - Results

$$\frac{kWh}{m^3_{effective}} = \frac{kWh}{m^3_{nominal}} \cdot \text{Charge Utilization}$$

Geometry Dependent

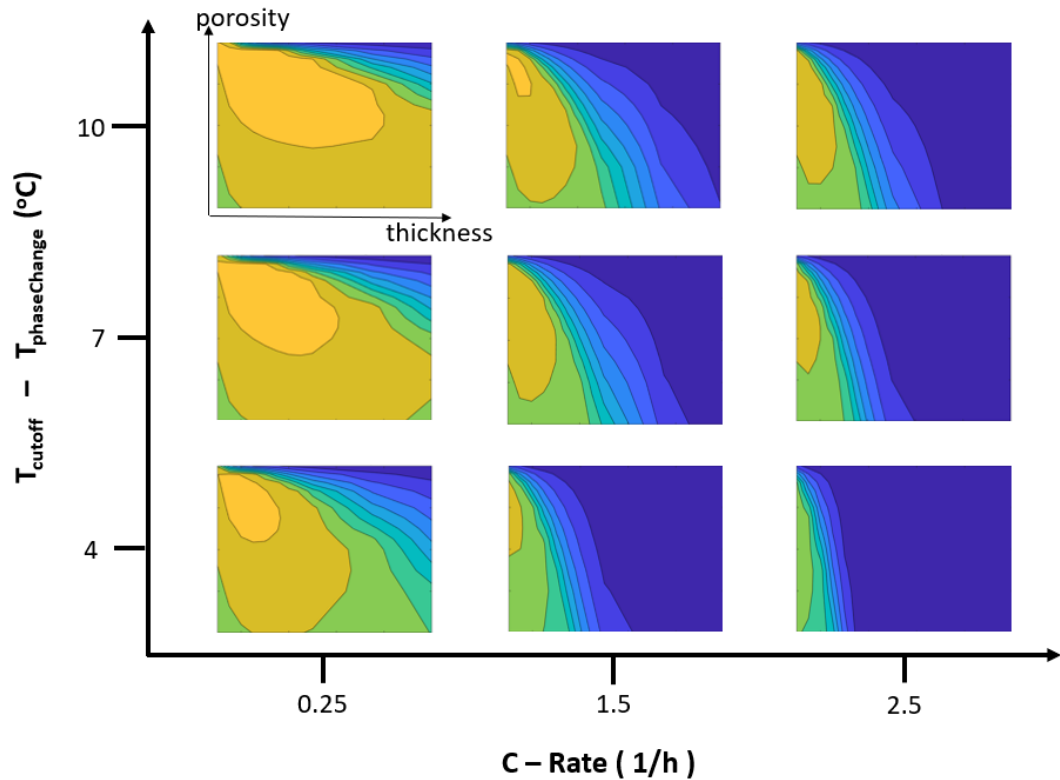


Geometry-Material-Operation Dependent

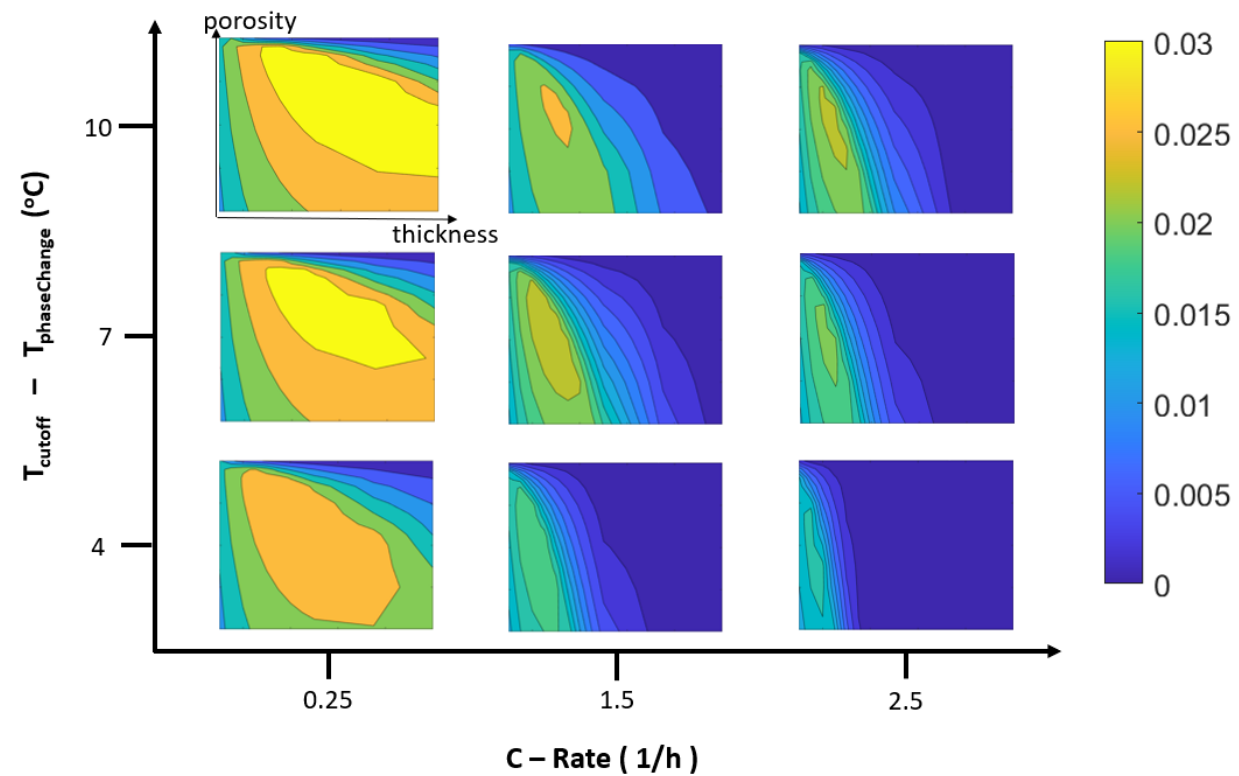


Finite Difference Model - Results

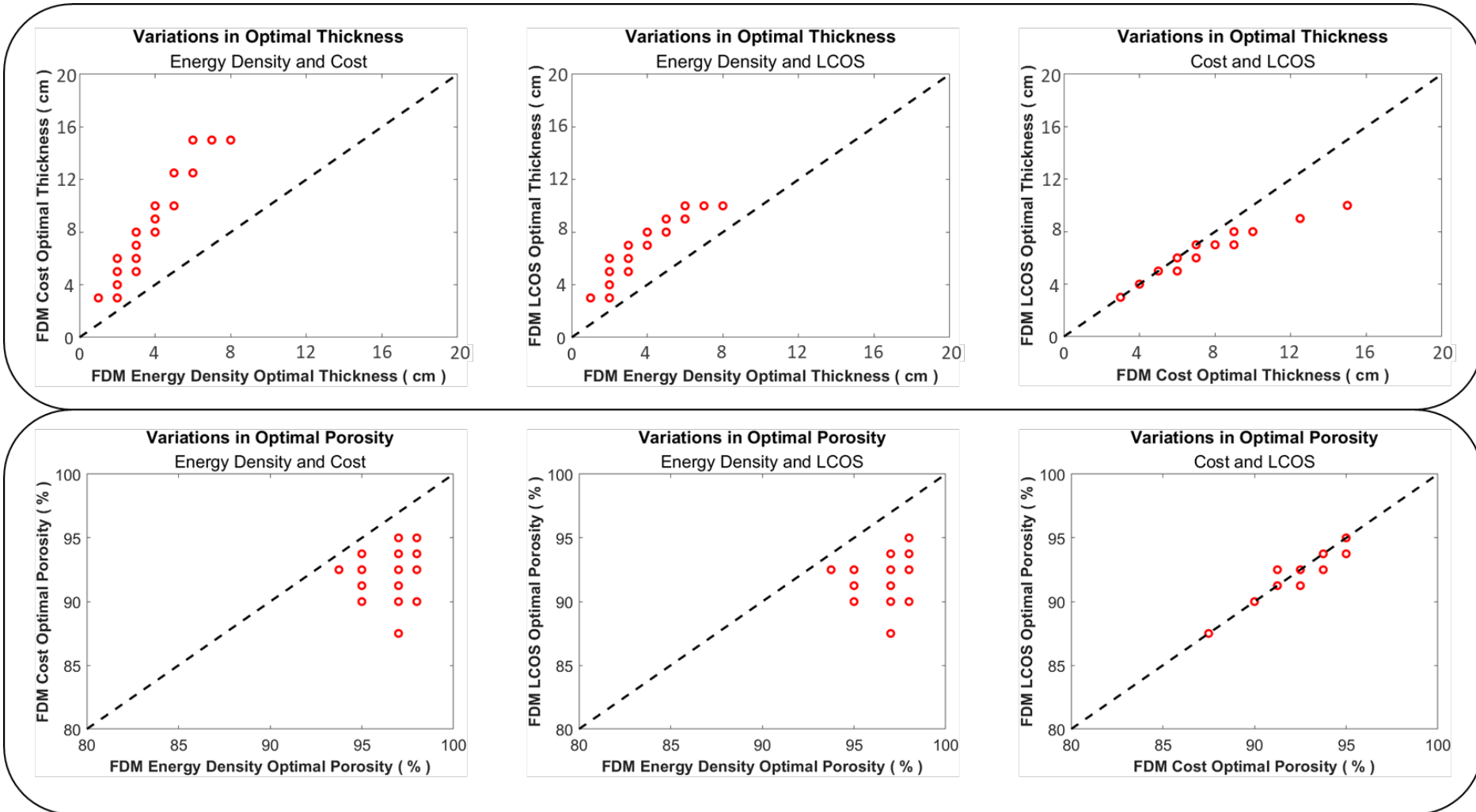
Effective Specific Energy Density (kWh/m³)



Inverse Capital Costs (kWh_T/\$)

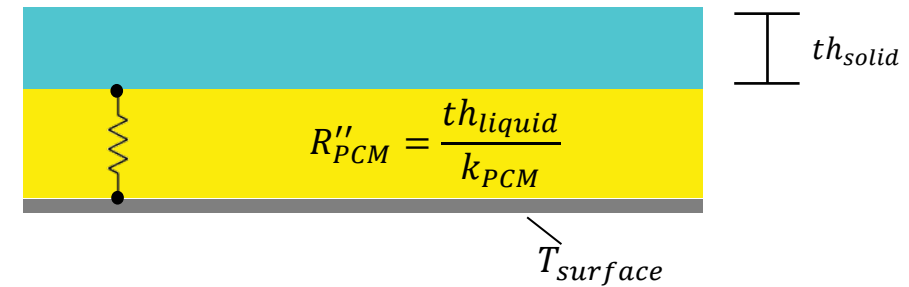


Finite Difference Model - Results



Parallel Phase Front Approximation Model

- Assumes that the melting front of the phase change composite is always parallel to the flow direction of the heat transfer fluid
- For a given driving temperature difference (cutoff temperature – Transition temperature) the maximum allowable phase change composite layer thickness needed for full charge utilization can be derived assuming a constant phase change composite thermal conductivity
- An effective charge utilization can be determined from relating the maximum thermal resistance of the PCM layer to the allowable thickness determined by the cutoff temperature for a given power delivery requirement



Target conditions:

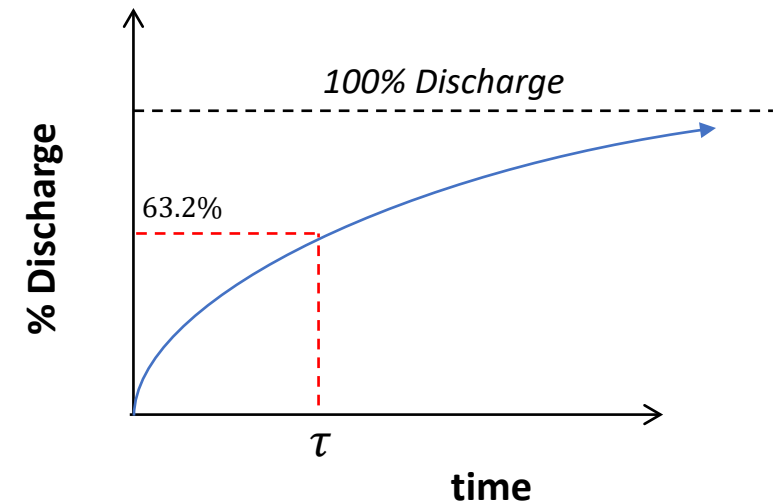
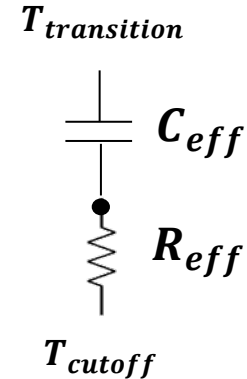
$$\dot{Q}_{target} = C_{rate} * Cap$$

Resistance when you reach cutoff:

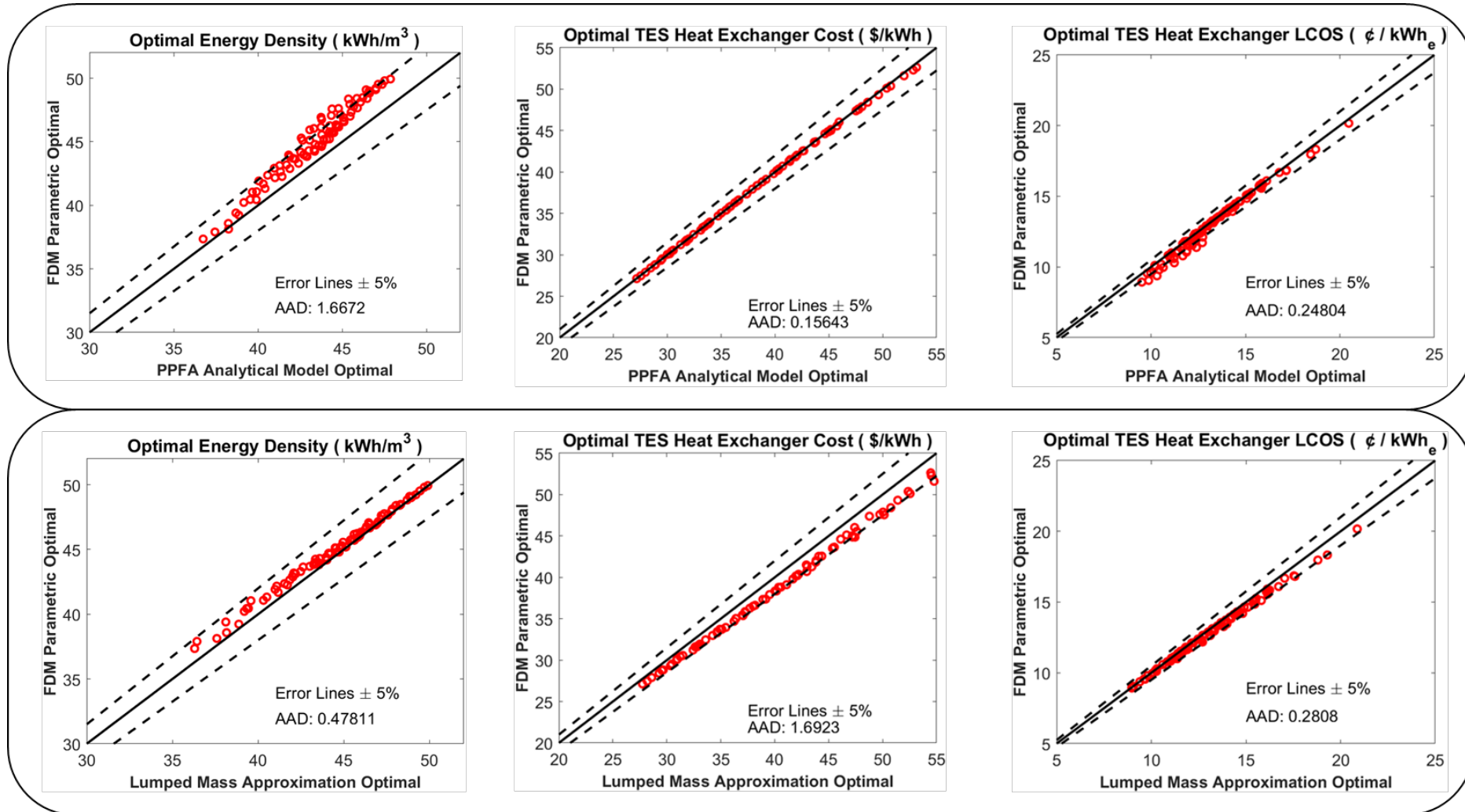
$$\dot{Q}_{target} = \frac{\bar{T}_{fluid,cutoff} - \bar{T}_t}{R_{conv} + R_{contact} + R_{cutoff}}$$
$$SOC_{cutoff} = 1 - \frac{R_{cutoff}}{R_{max}}$$

Lumped Mass Approximation Model

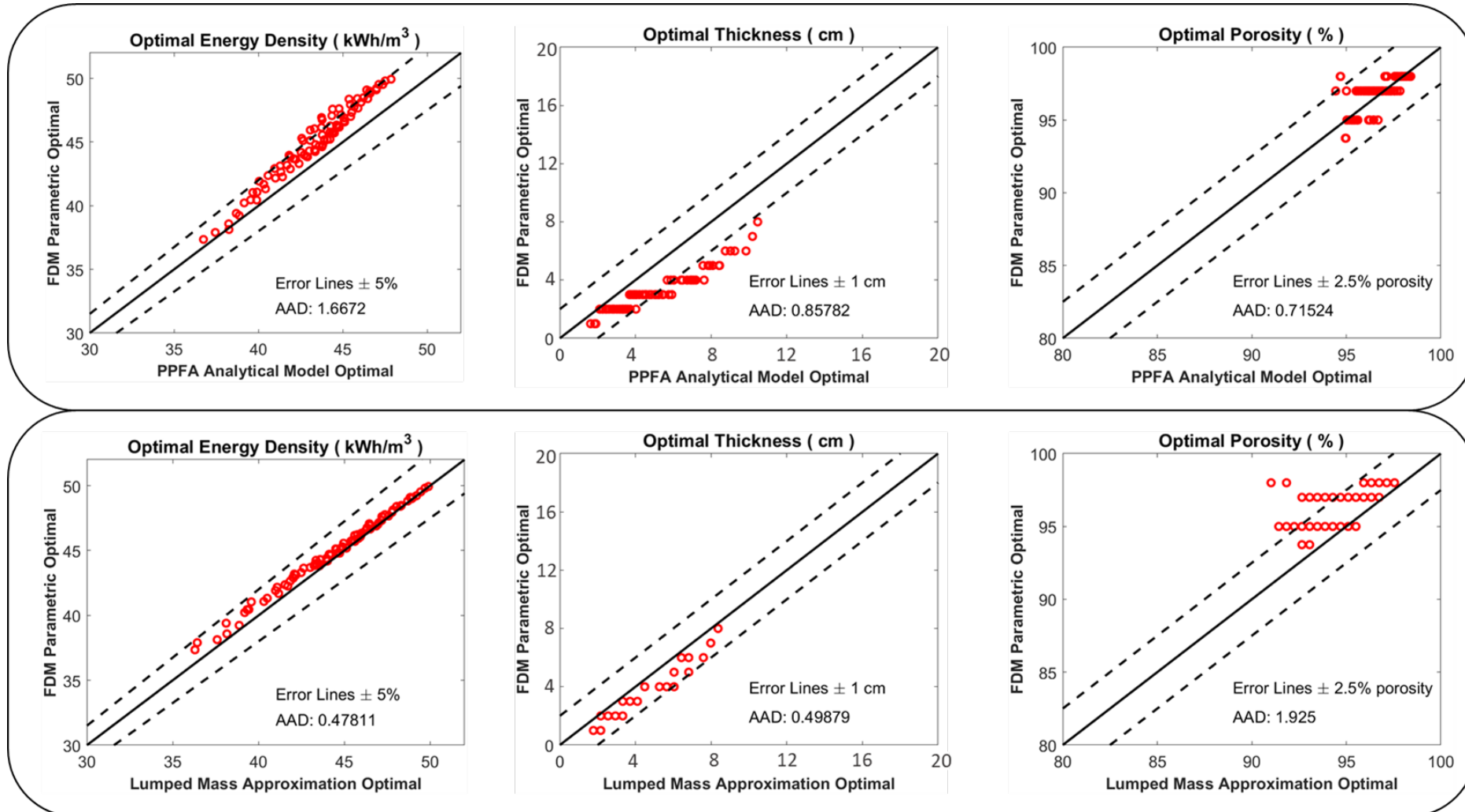
- Treats the thermal energy storage as an RC-circuit with an effective resistance and capacitance based on material and geometric properties
- Defines a time constant capturing the impact of the driving temperature difference
- Relates the time required for a nearly complete discharge (99%) to the time required for discharge determined by the target C-rate



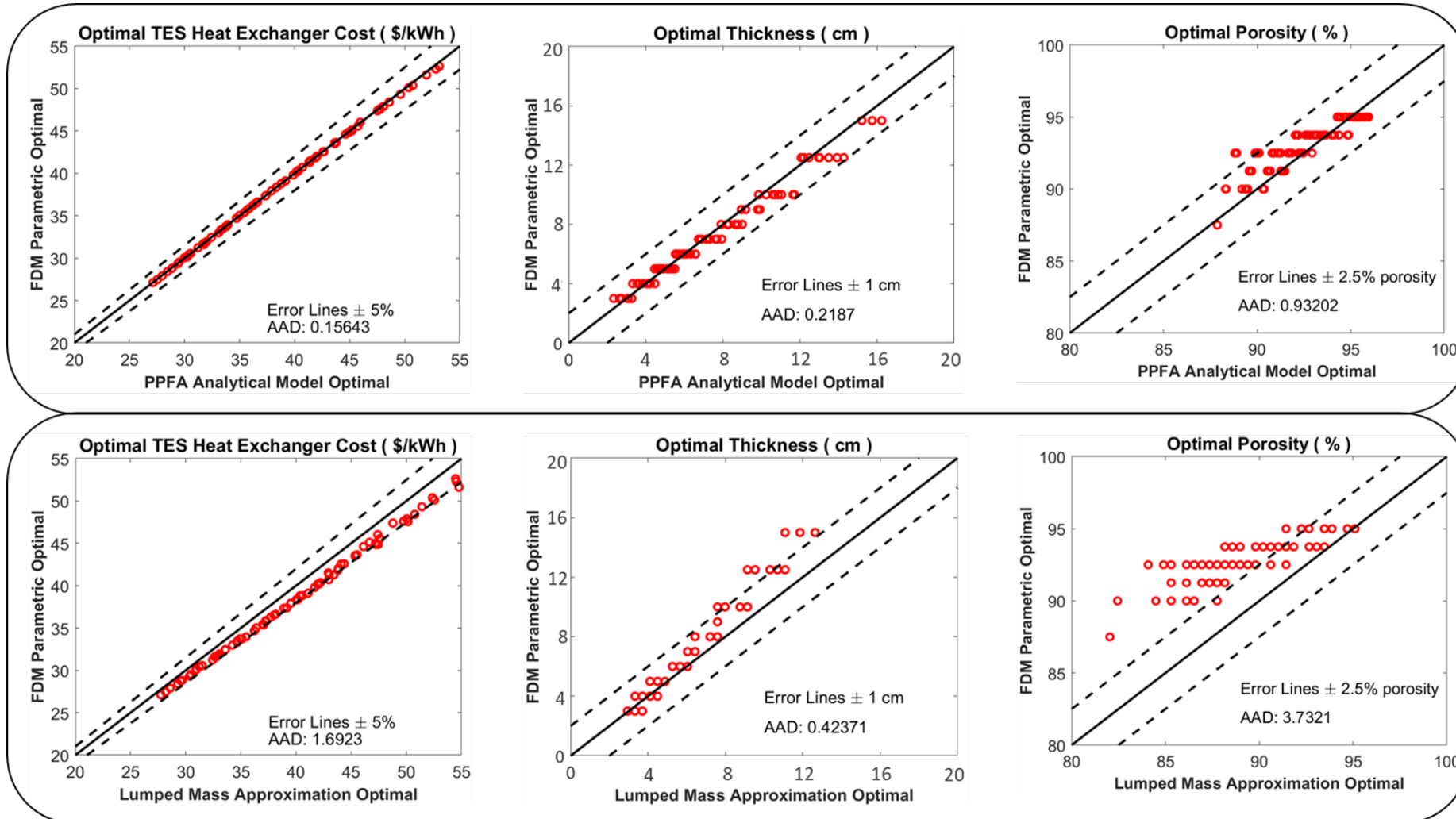
Model Comparisons



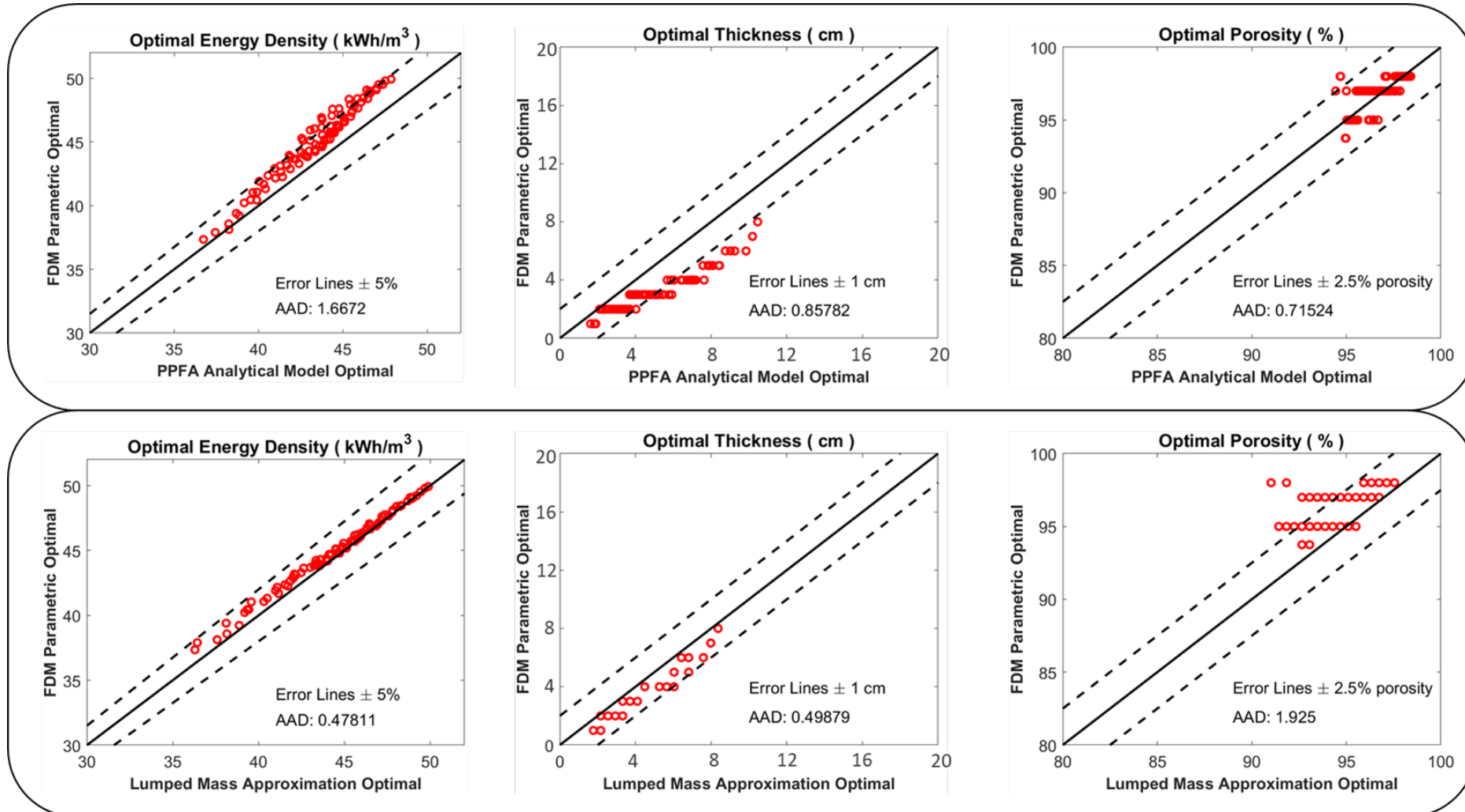
Model Comparisons



Model Comparisons



Model Comparisons



Conclusions

- As the C-Rate decreases and the driving temperature difference increases, the porosities and thicknesses needed to maximize energy density, minimize cost, or minimize LCOS will both increase
- Higher C-Rates and lower driving temperature differences necessitate thinner PCC layer thicknesses and lower porosity conductivity additives for optimizing thermal storage devices
- For the assumed material cost estimates, minimizing LCOS or energy-specific capital costs requires thicker PCC layers with less conductivity enhancing material compared to those needed to maximize the effective energy density
- Simplified models that incorporate elements of the Ragone framework were presented which can aid in accelerating the evaluation of thermal energy storage heat exchanger designs

Acknowledgments

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- Jason Woods - NREL

Bibliography

Woods et al., Rate capability and Ragone plots for phase change thermal energy storage. *Nat Energy* 6, 295–302 (2021)

James et al., Optimizing phase change composite thermal energy storage using the thermal Ragone framework. *J. Energy Storage* 56, 105875 (2022)

Questions

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