

From Chills to Thrills: Revolutionizing Energy Efficiency and Load Flexibility in Supermarket Refrigeration

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## **Project Overview**

#### Objective and Outcome

- Develop a next-generation self-contained, refrigerated open vertical display case (OVDC) that leverages radiation and thermal energy storage (TES) to:
  - Improve energy efficiency
  - Provide **demand flexibility**
  - Improve human comfort
  - Improve environmentally friendliness of refrigeration system

#### Team and Partners

- Emerson/Copeland
- National Renewable Energy Laboratory (NREL)
- NETenergy
- Albertsons
- ComEd

## Goal of the Presentation

- Overview of the problem statement
- Discuss the proposed innovation
- Overview of modeling approach and bench-scale experimentation
- Discuss results of the bench scale experiment to validate effectiveness of the proposed cooling system

#### Problem Statement: Supermarkets

- Supermarkets have the second highest EUIs in the commercial buildings sector:
  - Refrigeration accounts for roughly 50% of their electric energy.<sup>1</sup>
  - OVDCs comprise nearly 50% of total case line-ups.

Major fuels intensity by principal building activity, 2012–2018 thousand British thermal units per square foot



Data source: U.S. Energy Information Administration, *Commercial Buildings Energy Consumption Survey* \* Change is statistically significant at the 10% significance level.

\*\* Change is statistically significant at the 5% significance level.

Note: <u>Building Type Definitions</u> on the CBECS web page provides more information about the principal building activities.

#### Problem Statement: OVDC

- **Forced convection** used to cool refrigerated products results in large mass exchange with the surrounded space:
  - Air infiltration accounts for 80% of cooling load.
  - The spilled cold air adversely impacts human comfort.
  - Frost formation on evaporator restricts air flow and hampers heat transfer  $\rightarrow$  degrades energy efficiency.
  - Highly variable and non-uniform product temperature between shelves (up to 10°F).
- **Refrigeration heat** rejected into the sales floor **cannot be reclaimed** by heating systems.
- **Inability** to reliably **participate** in **demand response** (DR) events and load shaving/shifting strategies.



**Current State of Technology** 

## **Proposed Technology**

#### **Novel Features:**

- 1. Complete removal of air curtain system
- Hybrid radiant and convective cooling using R290.
- Thermal energy storage (TES).
- Water cooled integrated with building space and water heating.

#### Hybrid Cooled OVDC With an Integrated TES



#### Proposed Technology Schematics



#### Proposed Technology: R&D Targets

*Energy Efficiency:* 30% Improvement.

- by **reducing infiltration** load, frost formation, defrost cycles, and post-defrost pull-down loads **with radiant cooling** and minimizing forced convection by air curtain removal.
- by **integrating water-cooled condenser** with the building's heating systems to reduce temperature lift and recover waste heat.

**Demand Flexibility:** 80% peak kW reduction of display case over 2 hours.

• by adding **TES** 

Affordability: 3-year payback period.

• by **reducing operational costs** with energy savings and DR programs.

**Occupant Comfort:** 50% lower infiltration.

by reducing cold air spillage in shopping aisles.

#### Integrated Research Approach



#### Modeling Methodology

#### Modeling Approach based on Experimental Data

- Thermo-fluid heat extraction Resistive Network modeling of the OVDC
  - Validation using 3-D CFD modeling
  - Experimentation Data
- 2. Refrigeration system modeling
- 3. Thermal energy storage system modeling
- 4. Integrated system model
- Whole-building EnergyPlus<sup>®</sup> hourly simulation based on an actual Albertsons store





EnergyPlus Building Model

#### **Experimentation Methodology**

Baseline Fixture Bench Scale Fixture (Modified Baseline unit)

## **Baseline Experimentation**

- Laboratory characterization of a typical 8-ft long, 5-deck, selfcontained OVDC with a water-cooled condenser to calibrate the thermo-fluid model
  - ASHRAE 72 / AHRI 1200
- **117** channels of data collected every 1s over 24 hrs monitoring:
  - Temperatures: Air, refrigerant, water, product
  - Pressure: Refrigerant Suction & Discharge, Condenser water
  - Power: Total case and components
  - Mass Flow: Condenser water
  - Condensate Mass measured to calculate latent load



#### **Baseline System**



Baseline Display Case Cabinet: 8ft, 5-deck 13 ft3 gross volume

The Baseline refrigerated case is equipped with:

- water-cooled condensing unit
- advanced controls
- variable speed compressor
- electronic expansion valve



Electronic Expansion Valve



Advanced Controls



Liquid-cooled Condensing Unit with Variable Speed Compressor

## Bench Scale Hybrid Refrigerated Case

**Baseline case was modified with:** 

- 1. Complete elimination of the air curtain
- 2. Custom-fit radiant panels,
- 3. Variable-speed evaporator fan motors/controller
- 4. Back panel with modified perforations

#### Four Custom-Design Panels:

- bottom deck of the case (over the evaporator),
- two on the left and right wall around the shelves, and
- one on the canopy of the case.

\*perforated and replaced the DAG and enclosed canopy vent of the baseline

Each radiant panel contains a copper coil seated in a conductive trough to transfer heat from the low-emissivity panel surface





#### Bench Scale Refrigerated Case with Installed Radiant Panels



## **Experimentation Approach**

ASHRAE 72 and AHRI 1200
 Standards were used to maintain consistent conditions between tests

\*except used water bottles for filler material

 For the bench-scale validation, a chiller supplied the panels' coils with silicone coolant at controlled temperature and flow rate.



#### Instrumentation Used to Calculate Thermal Load of Panels:

- ST = surface thermocouples
- IPT = insertion probe thermocouple
- R = rheometer flowmeter
- CF = high-accuracy Coriolis flowmeter

### Experimental Setup – Bench Scale

#### **Case Inside Environmental Chamber**



#### **Radiant Cooling Loop**



A: port into/out of chamber
B: condensate drum and scale
C: flowmeters, pressure gauges, and thermocouples
D: chiller – serving radiant panels
E: control box for the chiller, valves, and flood switches to detect leaks.

### Results

#### **Modeling Results Informing** Hybrid Case Design

- Both CFD and EES models closely aligned with measured data:
  - Informed back panel air flow at 900 RPM and custom perforation for optimally-distributed air flow
  - Informed mean panel surface temperature of 32°F ۲





#### Mass Flow Distribution at BP & Canopy

Normalized Flow Rates w.r.t. RAG Flow

Convergence of Resistive Model with CFD and Experimentation Data

# Average product temperatures from EES, CFD, and experimental measurements:

	EES	CFD	Experimental
	(model approach 1)	(model approach 2)	Measures
Baseline system	38.4°F	38.2°F	38.45°F
Hybrid system (bench-scale)	39.2°F	39.1°F	39.01°F

#### Hybrid Radiant + Convective Cooling Experimental Results

- Varied fan speeds from 500 RPM (fan motor minimum) to 1600 RPM (baseline airflow)
- 900 RPM provided optimum BP airflow while maintaining mean product temp & minimizing Infiltration



Hybrid Radiant Cooling

Baseline w Air Curtain

### Conclusions

#### Conclusions

- Convergence of resistive model with CFD and experimentation data increased confidence in modeling platform, which will be leveraged to design the POC
- The hybrid cooling system:
  - Complied with FDA product temp targets. (39.0°F mean product temp at 900 RPM evaporator fan speed)
  - Reduced infiltration load by 96% (at 900 RPM evaporator fan speed) based on mass of condensate data)

#### Next Steps

- Use validated thermo-fluid model to identify further strategies for minimizing the cooling load of the case & developing specs
- Fabricate a hybrid proof-of-concept prototype fixture (including TES and advanced controls integration)
- Ascertain the energy efficiency benefits of the hybrid proof-ofconcept in a laboratory setting
- Assess the impacts of load flexibility strategies by leveraging the TES system in a laboratory setting

# Thank you!

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## Appendix

## **Thermo-Fluid Modeling**

- 1<sup>st</sup> approach (thermal resistive network model) used Engineering Equation Solver (EES): considers conduction, convection, and radiation heat transfer modes and predicts temperatures
- 2<sup>nd</sup> independent approach leveraged 3D CFD modeling.
- \*models reflect the geometry of an actual OVDC.

