

From Chills to Thrills: Revolutionizing Energy **Efficiency and Load Flexibility in** Supermarket Refrigeration

Ramin Faramarzi, P.E.

Alex Bulk

Sammy Houssainy, PhD

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Project Overview

• **Objective and Outcome**

- Develop a next-generation self-contained, refrigerated open vertical display case (OVDC) that leverages radiation and thermal energy storage (TES) to:
	- Improve **energy efficiency**
	- Provide **demand flexibility**
	- Improve **human comfort**
	- Improve **environmentally friendliness of refrigeration system**

Team and Partners

- Emerson/Copeland
- National Renewable Energy Laboratory (NREL)
- **NETenergy**
- Albertsons
- ComEd

Goal of the Presentation

- Overview of the problem statement
- Discuss the proposed innovation
- Overview of modeling approach and bench-scale experimentation
- Discuss results of the bench scale experiment to validate effectiveness of the proposed cooling system

Problem Statement: **Supermarkets**

- **Supermarkets** have the **second highest EUIs** in the commercial buildings sector:
	- **Refrigeration** accounts for roughly **50%** of their **electric energy.1**
	- **OVDCs** comprise nearly **50% of total** case line-ups.

Major fuels intensity by principal building activity, 2012-2018 thousand British thermal units per square foot

Data source: U.S. Energy Information Administration, Commercial Buildings Energy Consumption Survey * Change is statistically significant at the 10% significance level.

** Change is statistically significant at the 5% significance level.

Note: Building Type Definitions on the CBECS web page provides more information about the principal building activities.

Problem Statement: OVDC

- **Forced convection** used to cool refrigerated products results in large mass exchange with the surrounded space:
	- Air infiltration accounts for *80% of cooling load.*
	- The **spilled cold air** adversely **impacts human comfort.**
	- **Frost formation** on evaporator restricts air flow and **hampers heat transfer** \rightarrow *degrades energy efficiency.*
	- Highly variable and **non-uniform product temperature** between shelves (up to 10°F).
- **Refrigeration heat** rejected into the sales floor **cannot be reclaimed** by heating systems.
- **Inability** to reliably **participate** in **demand response** (DR) events and load shaving/shifting strategies.

Proposed Technology

Novel Features:

- 1. Complete removal of air curtain system
- 2. Hybrid radiant and convective cooling using R290.
- 3. Thermal energy storage (TES).
- 4. Water cooled integrated with building space and water heating.

Hybrid Cooled OVDC With an Integrated TES

Proposed Technology **Schematics**

Proposed Technology: R&D Targets

Energy Efficiency: 30% Improvement.

- by **reducing infiltration** load, frost formation, defrost cycles, and post-defrost pull-down loads **with radiant cooling** and minimizing forced convection by air curtain removal.
- by **integrating water-cooled condenser** with the building's heating systems to reduce temperature lift and recover waste heat.

Demand Flexibility: 80% peak kW reduction of display case over 2 hours.

• by adding **TES**

Affordability: 3-year payback period.

• by **reducing operational costs** with energy savings and DR programs.

Occupant Comfort: 50% lower infiltration.

• by **reducing** cold **air spillage** in shopping aisles.

Integrated Research Approach

Modeling Methodology

Modeling Approach based on **Experimental Data**

- 1. Thermo-fluid heat extraction Resistive Network modeling of the OVDC
	- Validation using 3-D CFD modeling
	- Experimentation Data
- 2. Refrigeration system modeling
- 3. Thermal energy storage system modeling
- 4. Integrated system model
- 5. Whole-building EnergyPlus® hourly simulation based on an actual Albertsons store

EnergyPlus Building Model

Experimentation Methodology

Baseline Fixture Bench Scale Fixture (Modified Baseline unit)

Baseline Experimentation

- Laboratory characterization of a typical 8-ft long, 5-deck, selfcontained OVDC with a water-cooled condenser to calibrate the thermo-fluid model
	- ASHRAE 72 / AHRI 1200
- 117 channels of data collected every 1s over 24 hrs monitoring:
	- Temperatures: Air, refrigerant, water, product
	- Pressure: Refrigerant Suction & Discharge, Condenser water
	- Power: Total case and components
	- Mass Flow: Condenser water
	- Condensate Mass measured to calculate latent load

Baseline System

Baseline Display Case Cabinet: 8ft, 5-deck 13 ft3 gross volume

The Baseline refrigerated case is equipped with:

- water-cooled condensing unit
- advanced controls
- variable speed compressor
- electronic expansion valve

Electronic Expansion Valve Advanced Controls

Liquid-cooled Condensing Unit with Variable Speed Compressor

Bench Scale Hybrid Refrigerated Case

Baseline case was modified with:

- **1. Complete elimination of the air curtain**
- **2. Custom-fit radiant panels,**
- **3. Variable-speed evaporator fan motors/controller**
- **4. Back panel with modified perforations**

Four Custom-Design Panels:

- bottom deck of the case (over the evaporator),
- two on the left and right wall around the shelves, and
- one on the canopy of the case.

***perforated and replaced the DAG and enclosed canopy vent of the baseline**

NREL | 15 Each radiant panel contains a copper coil seated in a conductive trough to transfer heat from the low-emissivity panel surface

Bench Scale Refrigerated Case with Installed Radiant Panels

Experimentation Approach

• ASHRAE 72 and AHRI 1200 Standards were used to maintain consistent conditions between tests

> *except used water bottles for filler material

• For the bench-scale validation, a chiller supplied the panels' coils with silicone coolant at controlled temperature and flow rate.

Instrumentation Used to Calculate Thermal Load of Panels:

- ST = surface thermocouples
- IPT = insertion probe thermocouple
- R = rheometer flowmeter
- CF = high-accuracy Coriolis flowmeter

Experimental Setup – Bench Scale

Case Inside Environmental Chamber Radiant Cooling Loop

A: port into/out of chamber B: condensate drum and scale C: flowmeters, pressure gauges, and thermocouples D: chiller – serving radiant panels E: control box for the chiller, valves, and flood switches to detect leaks.

Results

Modeling Results Informing Hybrid Case Design

- Both CFD and EES models closely aligned with measured data:
	- Informed back panel air flow at 900 RPM and custom perforation for optimally-distributed air flow
	- Informed mean panel surface temperature of 32°F

Mass Flow Distribution at BP & Canopy

Convergence of Resistive Model with CFD and Experimentation Data

Average product temperatures from EES, CFD, and experimental measurements:

Hybrid Radiant + Convective Cooling Experimental Results

- *Varied fan speeds from 500 RPM (fan motor minimum) to 1600 RPM (baseline airflow)*
- *900 RPM provided optimum BP airflow while maintaining mean product temp & minimizing Infiltration*

• Hybrid Radiant Cooling

• Baseline w Air Curtain

Conclusions

Conclusions

- Convergence of resistive model with CFD and experimentation data increased confidence in modeling platform, which will be leveraged to design the POC
- The hybrid cooling system:
	- Complied with FDA product temp targets. *(39.0°F mean product temp at 900 RPM evaporator fan speed)*
	- Reduced infiltration load by 96% *(at 900 RPM evaporator fan speed) based on mass of condensate data)*

Next Steps

- Use validated thermo-fluid model to identify further strategies for minimizing the cooling load of the case & developing specs
- Fabricate a hybrid proof-of-concept prototype fixture (including TES and advanced controls integration)
- Ascertain the energy efficiency benefits of the hybrid proof-ofconcept in a laboratory setting
- Assess the impacts of load flexibility strategies by leveraging the TES system in a laboratory setting

Thank you!

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Appendix

Thermo-Fluid Modeling

- **1st approach (thermal resistive network model) used Engineering Equation Solver (EES): considers conduction, convection, and radiation heat transfer modes and predicts temperatures**
- **2nd independent approach leveraged 3D CFD modeling.**
- *models reflect the geometry of an actual OVDC.

